

imaginary line horizontal connector of the car guides and are operatively connected by a shaft with the drive motor and a brake. The drive pulleys can be arranged between the drive motor and the brake on the shaft. The drive motor and brake can be mounted on a bracket fastened to the crossbeam.

5 An object of the present invention is to provide an elevator installation with a flexible arrangement of the drive motor. The drive motor is to be arranged freely and selectably substantially in the overall shaft region above the car and the counterweight. The drive motor is to be arranged in a space-saving manner and to be of small dimensions.

10 The present invention relates to an elevator installation with a car and a counterweight in a shaft. It comprises a drive motor mounted on a crossbeam. The crossbeam is fastened by way of each of two end regions at a respective counterweight guide and it is fastened by a center region to at least one car guide.

15 The two counterweight guides and one car guide span a substantially horizontal triangle in the shaft. The drive motor is of elongate and compact form. Advantageously the drive motor comprises two drive pulleys, which are arranged symmetrically to the left and right of a horizontal connector of the car guides.

20 By virtue of this arrangement, which is symmetrical in the triangle, of the guides, weight forces of the drive motor as well as bending moments arising during operation of the drive motor are effectively absorbed and conducted by way of the crossbeam and the guides to the shaft floor. The drive motor can be arranged freely and selectably on the area of this triangle substantially above the counterweight and/or substantially above the car. This flexibility with respect to the arrangement of the drive motor is made possible by the size and shape of the crossbeam and/or the number of deflecting rollers used
25 and/or the kind of drive means employed.

DESCRIPTION OF THE DRAWINGS

30 The above, as well as other advantages of the present invention, will become readily apparent to those skilled in the art from the following detailed description of a preferred embodiment when considered in the light of the accompanying drawings in which:

Fig. 1 is a schematic plan view of a triangular arrangement of guides of an elevator installation;

Fig. 2 is a perspective view of a first embodiment of the arrangement of a drive motor above the counterweight according to the present invention;

5 Fig. 3 is a schematic plan view of the arrangement of the drive motor shown in Fig. 2;

Fig. 4 is a schematic elevation view of the arrangement of the drive motor shown in Figs. 2 and 3;

Fig. 5 is a schematic plan view of a second embodiment of the arrangement of a
10 drive motor above the counterweight and/or the car according to the present invention;

Fig. 6 is a schematic elevation view of the arrangement of the drive motor shown in Fig. 5;

Fig. 7 is a schematic plan view of a third embodiment of the arrangement of a drive motor above the car according to the present invention;

15 Fig. 8 is a schematic elevation view of the arrangement of the drive motor shown in Fig. 7;

Fig. 9 is a schematic plan view of a fourth embodiment of the drive motor above the car according to the present invention; and

Fig. 10 is a schematic elevation view of the arrangement of the drive motor
20 shown in Fig. 9.

DESCRIPTION OF THE PREFERRED EMBODIMENT

Fig. 1 shows a schematic illustration of the triangular arrangement of guides **5, 5', 9, 9'** of an elevator installation in which a drive is to be arranged in accordance with the
25 present invention. The elevator installation is arranged in, for example, a substantially vertical shaft **10**. The shaft **10** has, for example, a rectangular cross-section with four walls. Substantially vertically arranged car guides **5, 5'** and counterweight guides **9, 9'** are fastened in the shaft. Two car guides guide a car **11** and two counterweight guides guide a counterweight **12**. The guides are fastened to adjacent walls. The two
30 counterweight guides **9, 9'** and a first car guide **5** are fastened to a first wall. A second car guide **5'** is fastened to a second wall. The second wall is disposed opposite the first wall. The first car guide **5** is arranged substantially centrally between the two

counterweight guides 9, 9'. The guides 5, 5', 9, 9' are formed of materials typically used in elevator installations, such as steel. The fastening of the guides to the walls is carried out by way of, for example, screw connections (not shown). With knowledge of the present invention, a person of ordinary skill in the elevator field could select other shaft
5 geometry with square, oval or round cross-section used.

The two counterweight guides 9, 9' and in each instance one of the two car guides 5, 5' form, in the shaft 10, substantially horizontal triangles T, T' respectively. The imaginary line horizontal connector between the two counterweight guides 9, 9' forms a first side of both of the triangles. The imaginary line horizontal connectors between each
10 counterweight guide and the car guide 5' form second and third sides of the triangle T. Advantageously, the horizontal connector between the counterweight guides 9, 9' is longer than each of the horizontal connectors to the car guide 5, so that the triangle T consisting of guides 9, 9', 5 at the first wall has an obtuse angle opposite the horizontal connector of the counterweight guides 9, 9'. The horizontal connector of the
15 counterweight guides 9, 9' is shorter than the horizontal connectors to the car guide 5', so that the triangle T' consisting of the counterweight guides 9, 9' at the first wall and the car guide 5' at the second wall has an acute angle opposite the horizontal connector of the counterweight guides 9, 9'. Advantageously the horizontal connector of the car guides 9, 9' intersects the horizontal connector of the counterweight guides 5, 5' substantially
20 centrally so that the triangles T, T' are substantially equilateral.

Figs. 2 to 10 show a drive motor 1, 1' with two drive pulleys 3, 3'. Advantageously the drive pulleys 3, 3' are operatively connected by way of a shaft 4, 4' with the motor 1, 1' and a brake 2, 2'. Advantageously, the motor and the brake are arranged at opposite ends of the shaft and the drive pulleys are arranged between motor
25 and brake in a central region of the shaft. A control and/or a transformer of the elevator installation is/are arranged in a switch box 6 advantageously at a wall of the shaft 10. In the embodiments according to Figs. 2 to 8 the drive motor 1 is gearless and of elongate form, i.e. the diameter of the drive motor as seen in a plane perpendicular to the axis of the shaft 4 is smaller than the length of the drive motor. In the embodiment according to
30 Figs. 9 and 10, the drive motor 1' is provided with a gear 40. In this form of embodiment as well, the drive motor 1' is of elongate form, i.e. the diameter of the drive motor as

seen in a plane perpendicular to the axis of the gear 40 is smaller than the length of the drive motor.

Advantageously, the two drive pulleys 3, 3' are arranged symmetrically to the left and right of a horizontal connector of the car guides 5, 5'. The drive pulleys 3, 3' are 5 advantageously smaller in diameter than the motor housing and/or the brake housing.

The drive motor 1, 1' arranged substantially horizontally in the shaft 10 drives the car 11 and the counterweight 12, which are interconnected by way of at least one drive means 19, 19' in the shaft. The drive means 19, 19' has two ends 18, 18'. The drive means 19, 19' is a cable and/or a belt of any form. The load-bearing regions of the drive 10 means consist of metal, such as steel and/or plastic material, such as aramide. The cable can be a single cable or multiple cable and the cable can also have an external protective sheathing of plastic material. The belt can be flat and externally unstructured and smooth or, for example, can be structured into wedge ribs or cogged belts. Advantageously two drive means are employed 19, 19'.

15 Each of the ends 18, 18' of the drive means 19, 19' is fixed to a shaft wall or a shaft ceiling or a car guide or a counterweight guide or a crossbeam 8 or to the car 11 or to the counterweight 12. Advantageously, the ends 18, 18' of the drive means are fixed by way of resilient intermediate elements for damping of solid-borne sound. The intermediate elements are, for example, spring elements which prevent transmission of 20 vibrations, which are perceived to be unpleasant, from the drive means to the shaft wall and/or the shaft ceiling and/or the car guide and/or the counterweight guide and/or the crossbeam and/or the car and/or the counterweight. Several exemplary forms of fixings of the ends of the drive means can be distinguished as follows:

- In the embodiment according to Figs. 3 and 4, the first end 18 of the drive 25 means is fastened to the shaft wall or the shaft ceiling or the car guide 5' and the second end 18' of the drive means is fastened to the shaft wall or the shaft ceiling or the crossbeam 8 or the car guide 5.

- In the embodiment according to Figs. 5 and 6 as well as Figs. 9 and 10, one or 30 both ends 18, 18' of the drive means is or are fastened to the shaft wall or the shaft ceiling or the car guide or the crossbeam.

- In the embodiment according to Figs. 7 and 8, the first end **18** of the drive means is fastened to the car **11** and the second end **18'** is fastened to the counterweight **12**.

According to the embodiments shown, two drive pulleys move two drive means
5 by way of friction couple. With knowledge of the present invention the expert can also use drive motors as well as drive means different from the examples. Thus, the expert can use a drive motor with only one drive pulley or with more than two drive pulleys. In addition, the expert can use a drive pinion, which drive pinion is in mechanically positive engagement with a cogged belt, as the drive means.

10 Several exemplary forms of suspensions can be distinguished:

- In the first embodiment according to Figs. 2 to 4, in the second embodiment according to Figs. 5 and 6 and in the fourth embodiment according to Figs. 9 and 10, the car **11** and the counterweight **12** are suspended with a 2:1 ratio. In the case of the 2:1 suspension of the car **11**, several deflecting rollers **13**, **13'**, **14**, **14'** are mounted on the car
15 **11**. In the case of the 2:1 suspension of the counterweight **12**, at least one deflecting roller **17**, **17'** is mounted at the counterweight **12**. Advantageously, in the first embodiment the drive motor **1** is arranged in a region substantially above the travel path of the counterweight **12**, i.e. in the vertical projection above the counterweight. In the second embodiment the drive motor **1** is advantageously arranged in a region
20 substantially completely above the travel path of the car **11**. Advantageously, in the fourth embodiment the drive motor **1'** is arranged in a region substantially above the travel path of the counterweight **12** and the car **11**, i.e. in the vertical projection above the counterweight and the car.

- In the third embodiment according to Figs. 7 and 8, the car **11** and the
25 counterweight **12** are suspended with a 1:1 ratio. Advantageously, the drive motor **1** in the third embodiment is arranged in a region substantially above the travel path of the car **11**, i.e. in the vertical projection above the car. The drive motor **1** in the third embodiment is advantageously arranged completely above the travel path of the car **11**.

Fig. 2 shows a perspective view of the first embodiment of the arrangement of a
30 gearless drive motor **1**. The drive motor **1** is mounted on the crossbeam **8** arranged substantially horizontally in the shaft **10**. The crossbeam **8** is, for example, an elongate rectangle of proven materials, such as steel. In this first embodiment, the crossbeam **8** is

fastened to the counterweight guides 9, 9' and to the car guide 5 at the first wall of the shaft 10. Advantageously the crossbeam 8 is fastened by way of two end regions to the counterweight guides and by way of a center region to the car guide. The fastening of the crossbeam to these three guides takes place in the three fastening regions by way of, 5 for example, screw connections.

Advantageously the drive motor 1 is mounted on the crossbeam 8 indirectly by way of a bracket 7. The bracket 7 is advantageously mounted at the center region of the crossbeam 8. For example, the bracket 7 is mounted on the crossbeam 8 by way of feet 7.5, 7.6. The bracket 7 consists of, for example, a flat-edge or square member of proven 10 materials, such as steel, and is mounted on the crossbeam 8 by way of, for example, screw connections. Advantageously the drive motor 1 is fastened to the bracket 7 by way of a motor housing and a brake housing. The motor housing is advantageously fastened to a first bracket mount 7.1 and the brake housing to a second bracket mount 7.2. The two bracket mounts 7.1, 7.2 are connected together by way of, for example, 15 struts 7.3, 7.4 to be stiff in bending with respect to the axis of the shaft 4. Advantageously, the bracket mounts 7.1, 7.2 embrace, at least regionally, boundaries of the motor housing or the brake housing. For example, the bracket mounts 7.1, 7.2 embrace end faces of the motor housing or the brake housing. Advantageously, the motor 1 and the brake 2 are arranged in a region substantially outside an enclosure of the 20 bracket 7, whilst the drive pulleys 3, 3' are arranged in a region substantially within the enclosure of the bracket 7.

The crossbeam 8 is fastened at least to the apices of the triangle T. Advantageously, the crossbeam 8 rests by two end regions on the counterweight guides 9, 9' and it bears by the center region laterally against the car guide 5.

25 Distinction can be made between several exemplary embodiments of crossbeam fastenings:

- In the embodiment according to Figs. 2 to 4 - where the drive motor is arranged in a region substantially above the travel path of the counterweight - the crossbeam 8 is fastened to the counterweight guides 9, 9' and to the car guide 5 at the first wall, which 30 lies closest to the counterweight guides 9, 9' as well as the car guide 5. The crossbeam has the form of a rectangle.

- In the embodiments of Figs. 2 to 10 - where the drive motor 1, 1' is arranged in a region substantially above the travel path of the counterweight or the car - the crossbeam 8 is fastened to the counterweight guides 9, 9', to the car guide 5 of the first wall and/or to the car guide 5' of the second wall. The crossbeam 8 has, in the examples 5 of the embodiments according to Figs. 5 to 8, a triangular form with straight or curved sides and in the example of the embodiment according to Figs. 9 and 10 a T-shape.

The bracket 7 and the drive pulleys 3, 3' are advantageously arranged in a central region of the triangles T, T'. Advantageously, the bracket 7 is mounted at the center region of the crossbeam 8. For example, in the first embodiment according to Fig. 2 the 10 feet 7.5, 7.6 of the bracket 7 are mounted at the crossbeam 8 on opposite sides of the car guide 5 and substantially equally spaced from the car guide. For example, the drive pulleys 3, 3' are arranged on the shaft 4 on opposite sides of the car guide 5 and substantially equally spaced from the car guide.

The drive motor 1, 1' can thus be freely selectably arranged on the area of the 15 triangles T, T' substantially above the counterweight and/or substantially above the car. By virtue of this arrangement, which is symmetrical in the triangles T, T', of the guides, weight forces of the drive motor as well as bending moments arising during operation of the drive motor are effectively absorbed by, for example, the bracket 7 and conducted by way of the crossbeam 8 and the guides to the shaft floor. The guides are supported, for 20 example, by way of foot plates (not shown) on the shaft floor.

For example, in the embodiment according to Fig. 2 the first bracket mount 7.1 absorbs drive forces emanating from the motor 1 and the second bracket mount 7.2 absorbs braking forces emanating from the brake 2. In addition, the two bracket mounts 7.1, 7.2, absorb forces emanating from the drive pulleys 3, 3'. Advantageously, the two 25 drive pulleys 3, 3' are arranged symmetrically to the left and right of the horizontal connector of the car guides 5, 5'.

In addition, in the embodiments according to Figs. 5 to 8 - where at least one deflecting roller 15, 15' and 16, 16' is provided in the region above the counterweight 12 and/or substantially above the car 11 - forces emanating from this deflecting roller can be 30 absorbed by the crossbeam 8. Advantageously, this deflecting roller is fastened to the crossbeam 8 or to the bracket 7. Pairs of the deflecting rollers 15, 15', 16, 16' are advantageously arranged symmetrically to the left and right of the horizontal connector

of the car guides 5, 5'. Due to the number and position of the deflecting rollers, a flexibility in the arrangement of the drive motor 1 on the area of the triangles T, T' is made possible. In particular, a high degree of utilization of the shaft volume can be realized wherein a dead volume is largely avoided. In addition, the arrangement of the
5 drive motor can be flexibly adapted to predetermined shaft relationships even in the case of modernizing operations, which flexibility thus enables use of standard parts and avoids costly special solutions.

In accordance with the provisions of the patent statutes, the present invention has been described in what is considered to represent its preferred embodiment. However, it
10 should be noted that the invention can be practiced otherwise than as specifically illustrated and described without departing from its spirit or scope.